



# First Workshop ARENHA project: “Introduction to novel technologies related to ammonia-based energy storage”

## Direct use of ammonia for mobility (ICE)

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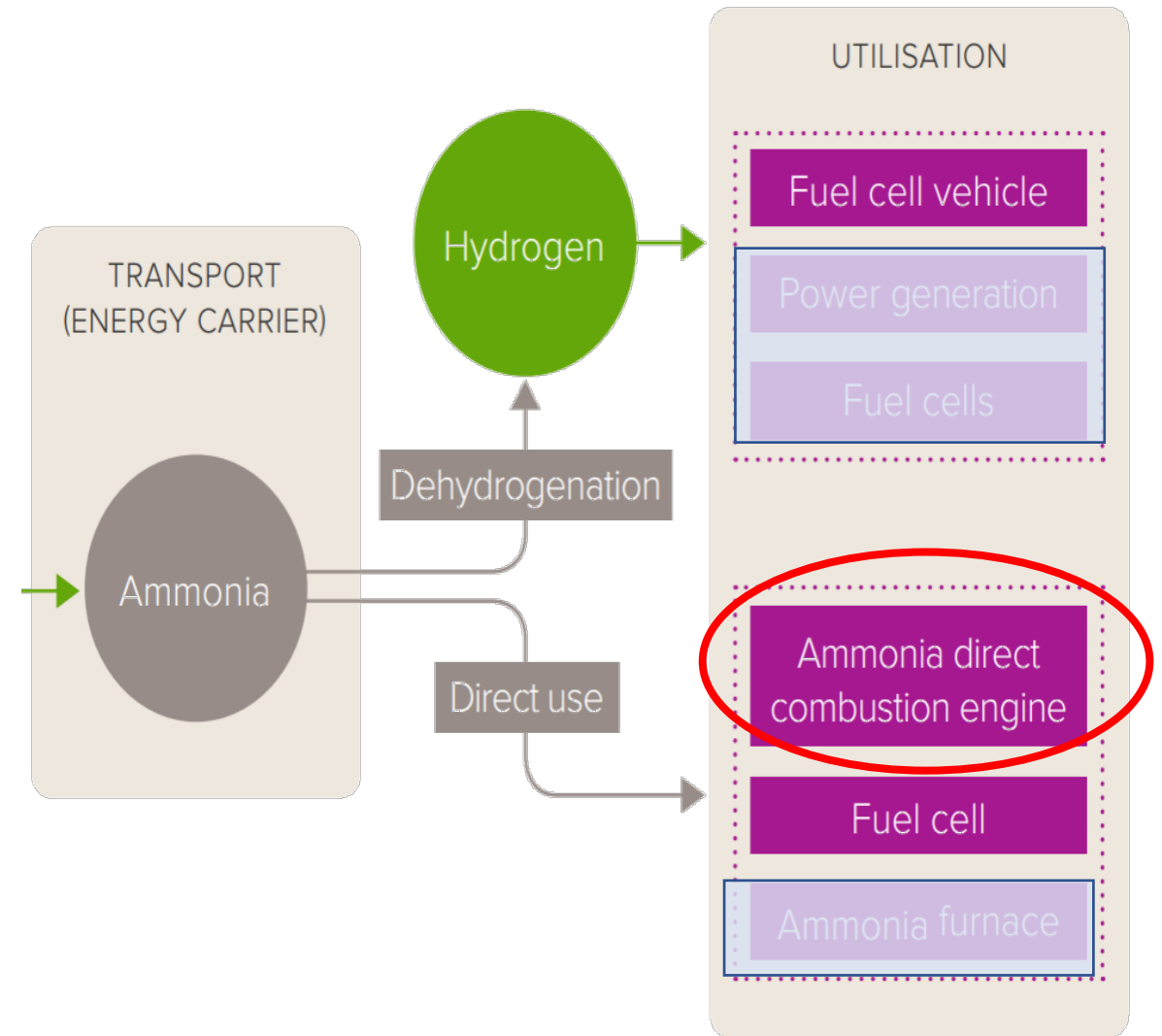
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# I. How extract the energy content in ammonia ?

## 3 solutions

- ❑ To decompose ammonia by means of new optimized reformers to recover hydrogen
- ❑ To use ammonia directly
  - ❑ in fuel cells
  - ❑ In combustion systems such as turbines or internal combustion engines.

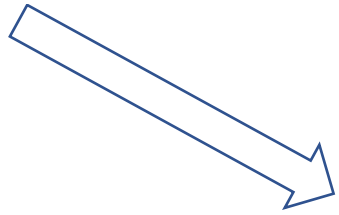




# I. $\text{NH}_3$ as fuel for vehicles = an old story

areNH<sub>3</sub>a

60s : theoretical studies, 'Research Engine' CFR studies (USA)



2007-2012 :  
Michigan  
University  
50% $\text{NH}_3$ /Gasoline  
3 800 km



2012-2015 :  
KIER, Korea  
Dual Fuel  
gasoline or Diesel  
until 80%  $\text{NH}_3$



2013  
Marangoni Toyota  
GT-86 Eco-  
Explorer,



1940s  
Belgium  
 $\text{NH}_3$ /Coal gas  
10 000 miles



1933 : Nork Hydro  
truck ran with  
ammonia

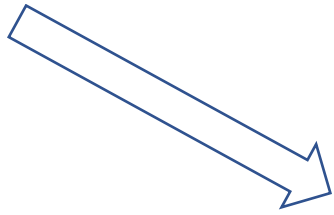
First ARENHA Workshop, ENGIE Lab CRIGEN (April 7<sup>th</sup>, 2022)  
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# I. NH<sub>3</sub> as fuel for vehicles = an old story

60s : theoretical studies, 'Research Engine' CFR studies (USA)



1933 : Nork Hydro truck ran with ammonia



1940s  
Belgium  
NH<sub>3</sub>/Coal gas  
10 000 miles



2013  
Marangoni Toyota  
GT-86 Eco-  
Explorer,



2012-2015 :  
KIER, Korea  
Dual Fuel  
gasoline or Diesel  
until 80% NH<sub>3</sub>



2013  
Università di Pisa  
H<sub>2</sub> reformer



2020  
Hydrofuel  
Project  
(Ontario Univ.)



2018-2021  
C-Free Run project,  
Hydrogen Engine Center  
(Iowa) Ford 460 (9.4 l / CR  
13.5) max RPM 2500  
Start and stop engine on  
hydrogen, when warm run  
on 85-90% ammonia and  
10-15% hydrogen




ACTIVATE  
norwegian project,  
Silesian University  
project 2022-  
2024



# I. Global ammonia combustion characteristics

areNH<sub>3</sub>a

	Hydrogen	Methane	Methanol	Gasoline	Diesel Fuel	Ammonia	consequences
Low Heating Value (MJ/kg)	120	49	19.9	44	45	18,6	Compensated by air/fuel ratio
Air/Fuel ratio at stoichiometry (kg/kg)	34.2	17.65	6.46	14.6	14.6	6.06	High fuel consumption
Flammability limit in air (vol.%)	4.5-75	5-15	6.7-36	1.3-7.6	1-6	15-30	Low risk
Laminar flame speed (cm/s)	210	38	40	~40		7	Difficult propagation
Auto-ignition Temperature(°C)	537	595	465	275	225	651	Difficult ignition
Octane Number (-)	>120	120	109	88-98		>120	Low knock occurency ?
Adiabatic flame temperature (°C)	2519	2326	2228	2392		2107	Colder flame
Quenching distance (mm)	0.64	2		3	7	7	Lower heat wall loss



Combustion and Performances in SI engines				
Minimum H <sub>2</sub> for combustion stability		Efficiency		Output energy
Between 5-10% in vol		Higher for ER>=1		Less than gasoline at low and partial load
Amount needed decreases with load increase (full load: 0%)		Higher than gasoline		Increase with CR or boosted pressure
slight effect of engine speed		Decrease with H <sub>2</sub> increase		But only from 2000
Pollutant Emissions before any aftertreatment device				
	ER decrease (lean)	ER increase (rich)	H <sub>2</sub> increase	Load
NO <sub>x</sub> (ppm)	++ maximum > gasoline	--	+	slight increase but no universal trend
Unburnt NH <sub>3</sub>	--	++	-- But H <sub>2</sub> at exhaust	no universal trend

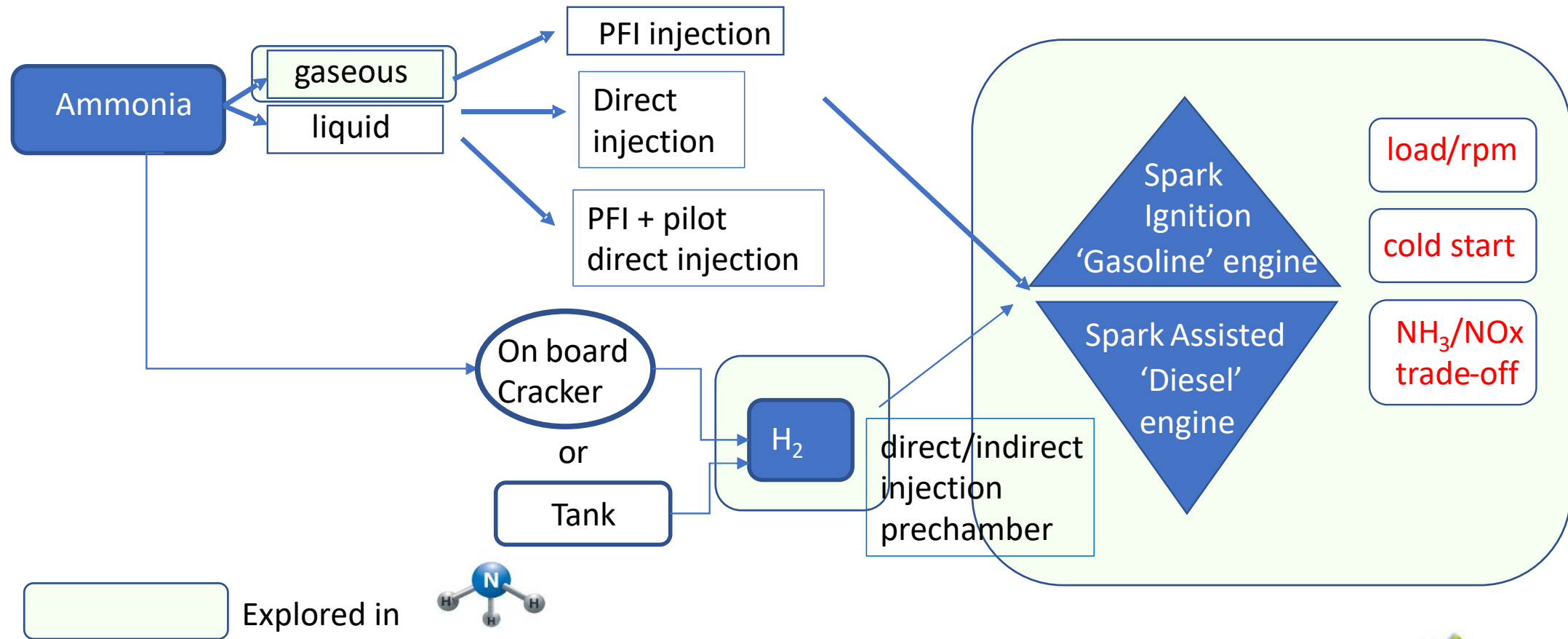
But only from 2000 to 4000 rpm in small engine



# I. Best possibilities to use ammonia only in ICE

Difficulties to auto-ignite  $\text{NH}_3$  :

- spark ignition mode : optimum





## 2. Impact of engine architecture



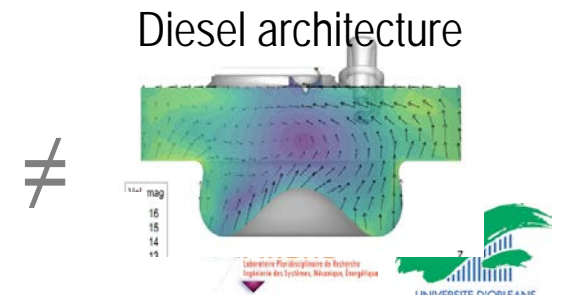
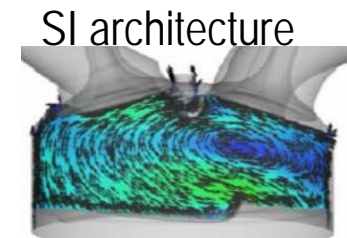
- Objectives : assessment of combustion stability, efficiency, pollutants for pure NH<sub>3</sub>
  - Identification of H<sub>2</sub> requirement
  - Specificity of 'cold start' conditions (650 rpm)
  - Identification of limits and tradeoffs (NO<sub>x</sub> versus NH<sub>3</sub>)



## 2. Impact of engine architecture

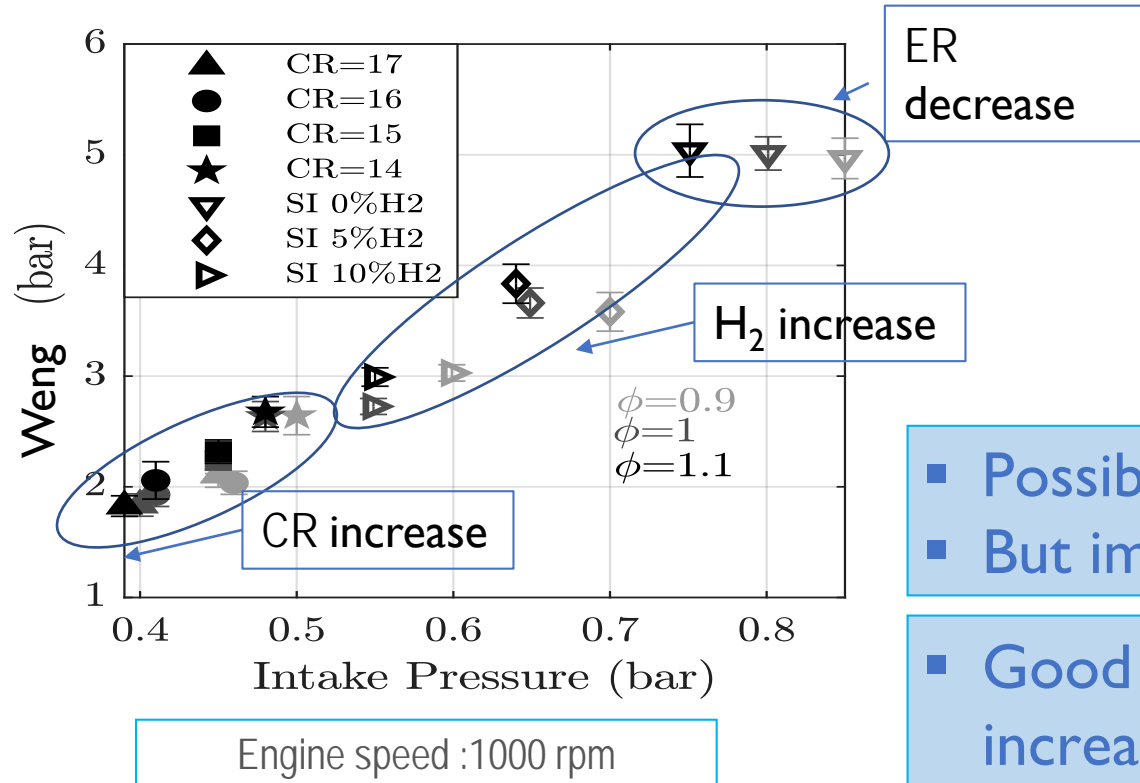


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  - Specificity of 'cold start' conditions (650 rpm)
  - Identification of limits and tradeoffs (NOX versus NH<sub>3</sub>)
- Different engines designs :
  - 2 standard engine : gasoline and diesel (but in single cylinder mode) :
    - SI engine = 'current' EP6
      - regular Compression Ratio
    - SA Diesel engine = 'current' DV6 + **spark plug instead of fuel injector**
      - High Compression ratio : **better for Ignition and Flame propagation**
  - I research large stroke engine SI
    - SI engine with high CR



## 2. Impact of engine architecture

❑ Solution : Increase the CR to reach 'cold start' conditions



Engine Type	Gasoline engine PSA EP6DT	Diesel engine PSA DV6
Displacement Volume $V_{cyl}$	400 cm <sup>3</sup>	400 cm <sup>3</sup>
Compression Ratio	10.5	14 to 17
Valves	4	2
Tumble ratio	2.4	
Swirl ratio		2

- Possible to run without H<sub>2</sub> even with standard SI
- But impossible to reach stable conditions without H<sub>2</sub>

- Good improvement of NH<sub>3</sub> combustion with CR increase despite of flow field
- No H<sub>2</sub> needs
- Extension of low load limits
  - lower limit with slightly rich

## 2. Impact of engine architecture

❑ Solution : Increase the CR to reach ‘cold start’ conditions

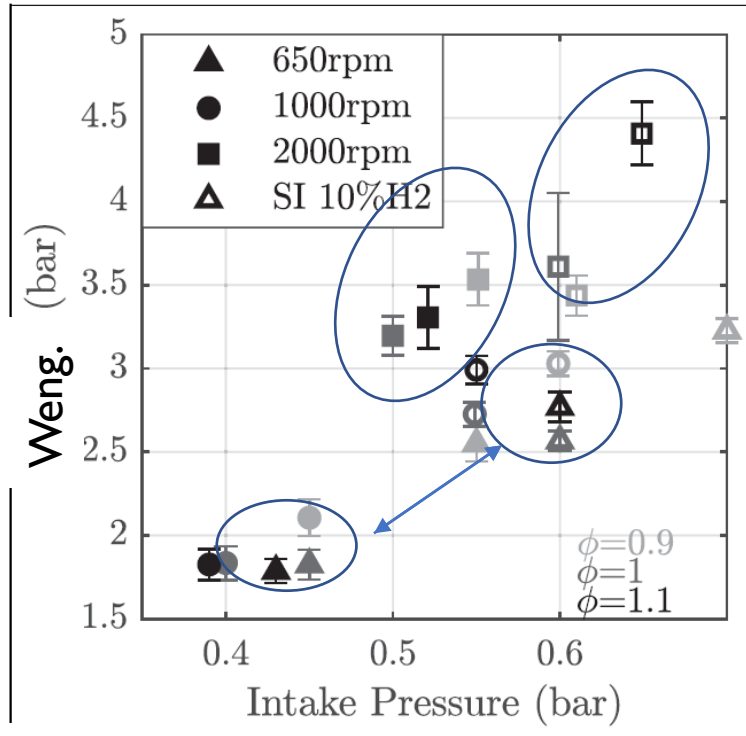
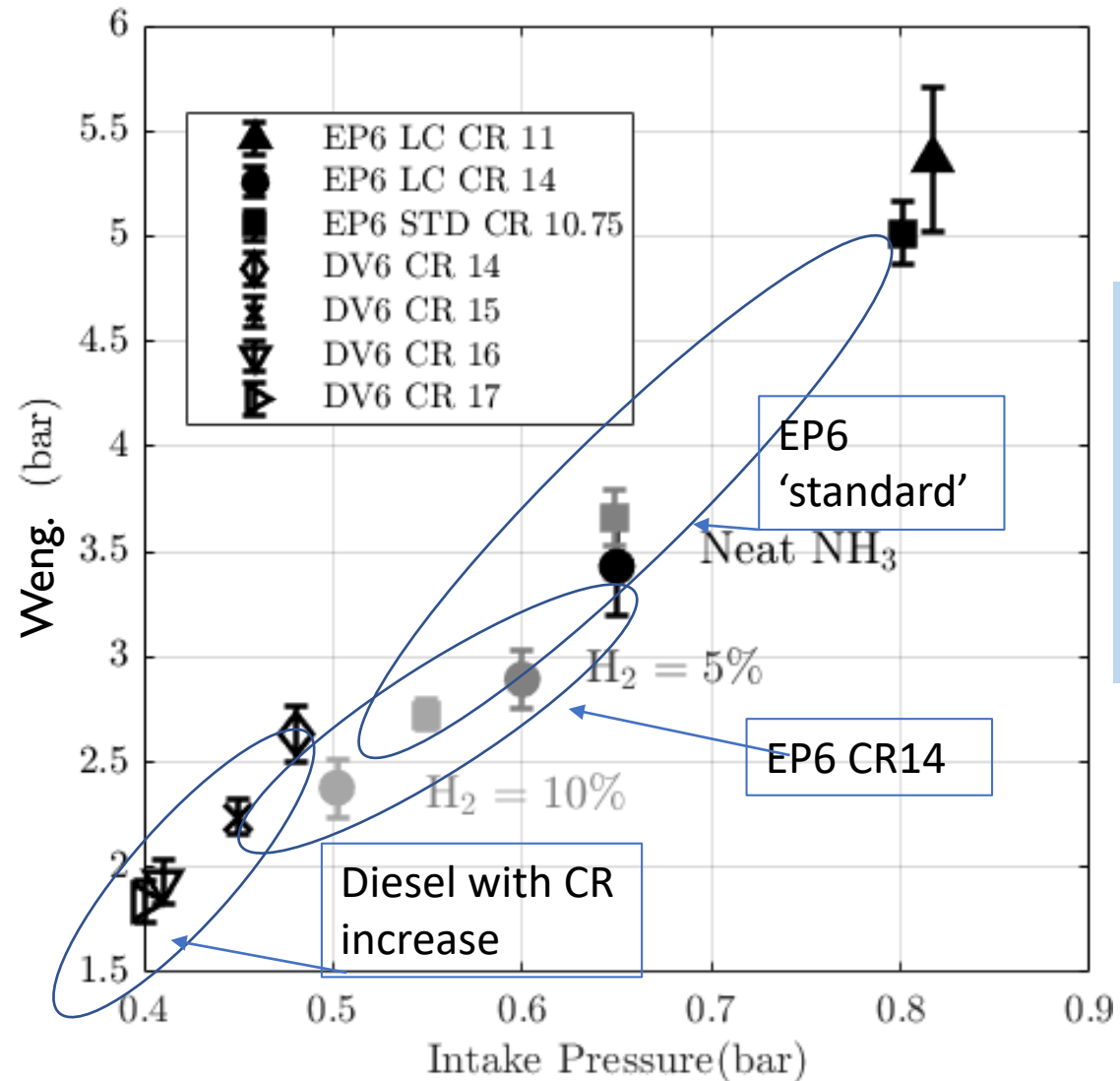
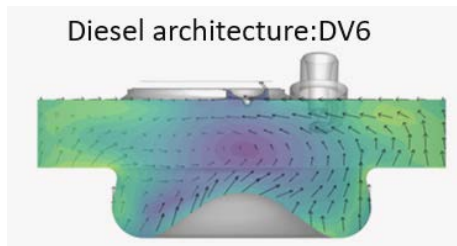
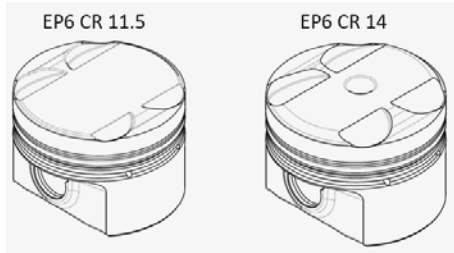


Figure 12. Minimum IMEP versus minimum intake pressure for three engine speeds – CR = 17:1,  $T_{in} = 50^{\circ}\text{C}$ . Previous data in SI engine with 10%  $\text{H}_2$ .

Engine Type	Gasoline engine PSA EP6DT	Diesel engine PSA DV6
Displacement Volume $V_{cyl}$	400 $\text{cm}^3$	400 $\text{cm}^3$
Compression Ratio	10.5	14 to 17
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- Extension of low load limits
  - lower limit with slightly rich
  - 1.7 b IMEP (as Koike et al. with Reformer)
  - CR 17, 650 rpm
  - Even at 2000 rpm, stability and limit improvement

## 2. Impact of engine architecture



Example at 650 rpm :

- H<sub>2</sub> required with for research SI engine even at CR14 !
- Lowest IMEP in SAD engine, even without H<sub>2</sub>

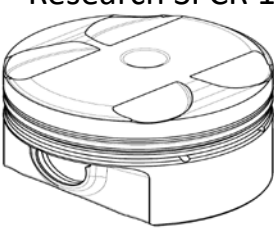
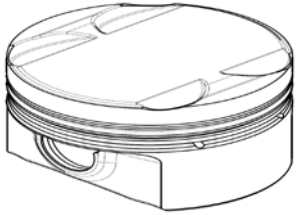


areNH<sub>3</sub>a

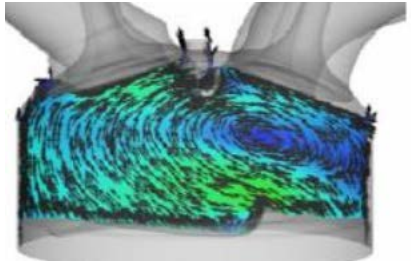
# 3. Consequence on emissions

Research SI CR 11.5

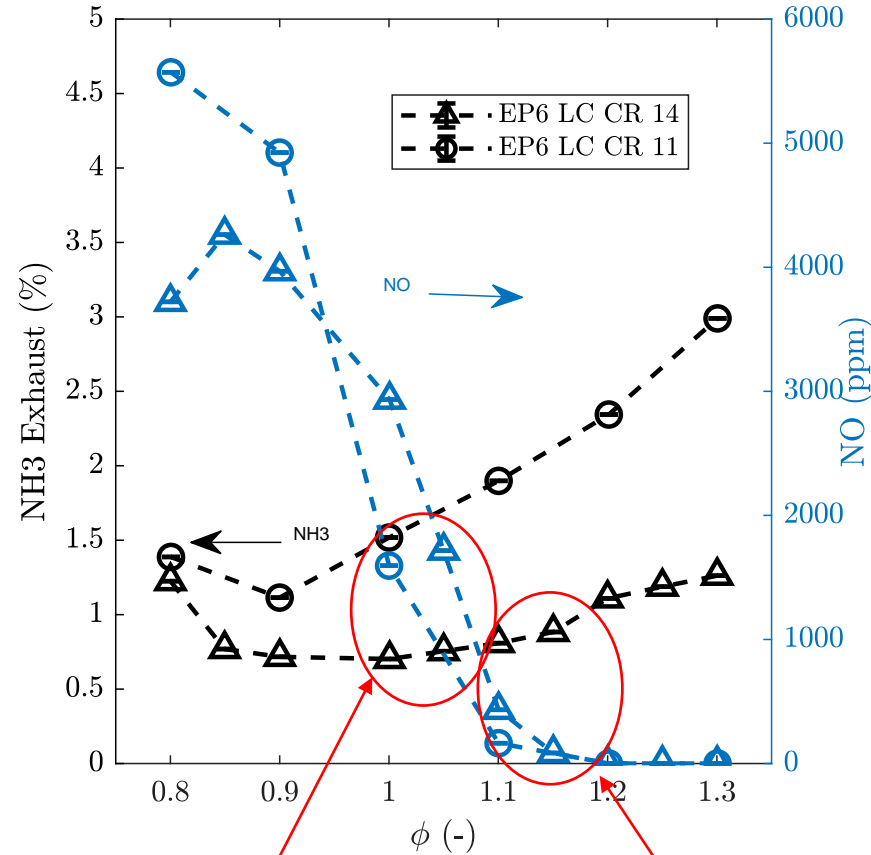
Research SI CR 14



4 valves, 0.4 l

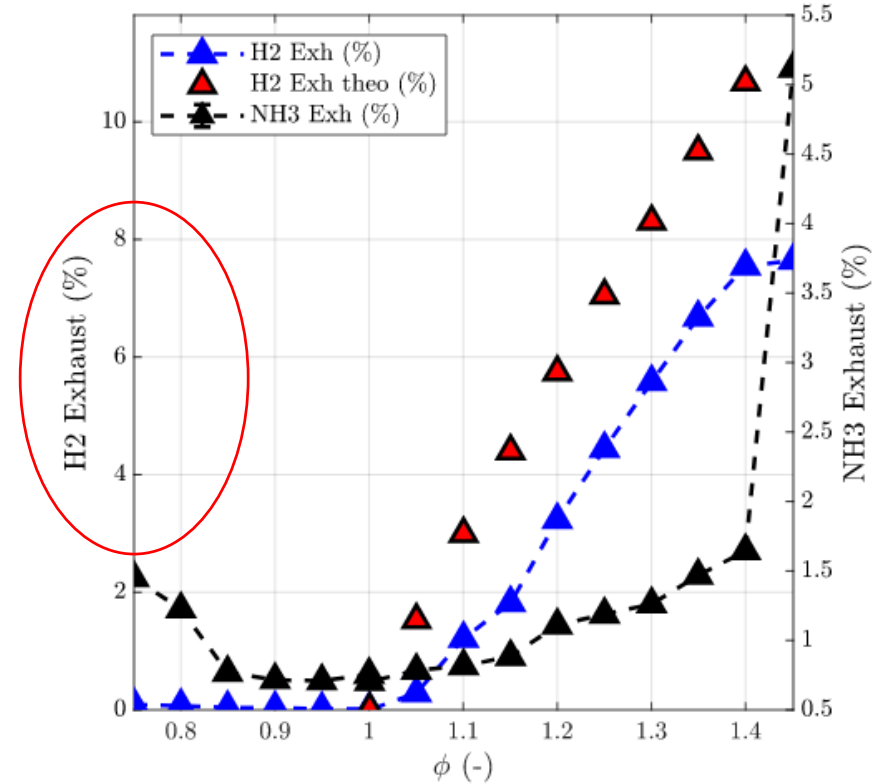


1000 rpm, intake pressure = 1 bar,  
IMEP = 8 b, NH<sub>3</sub> only

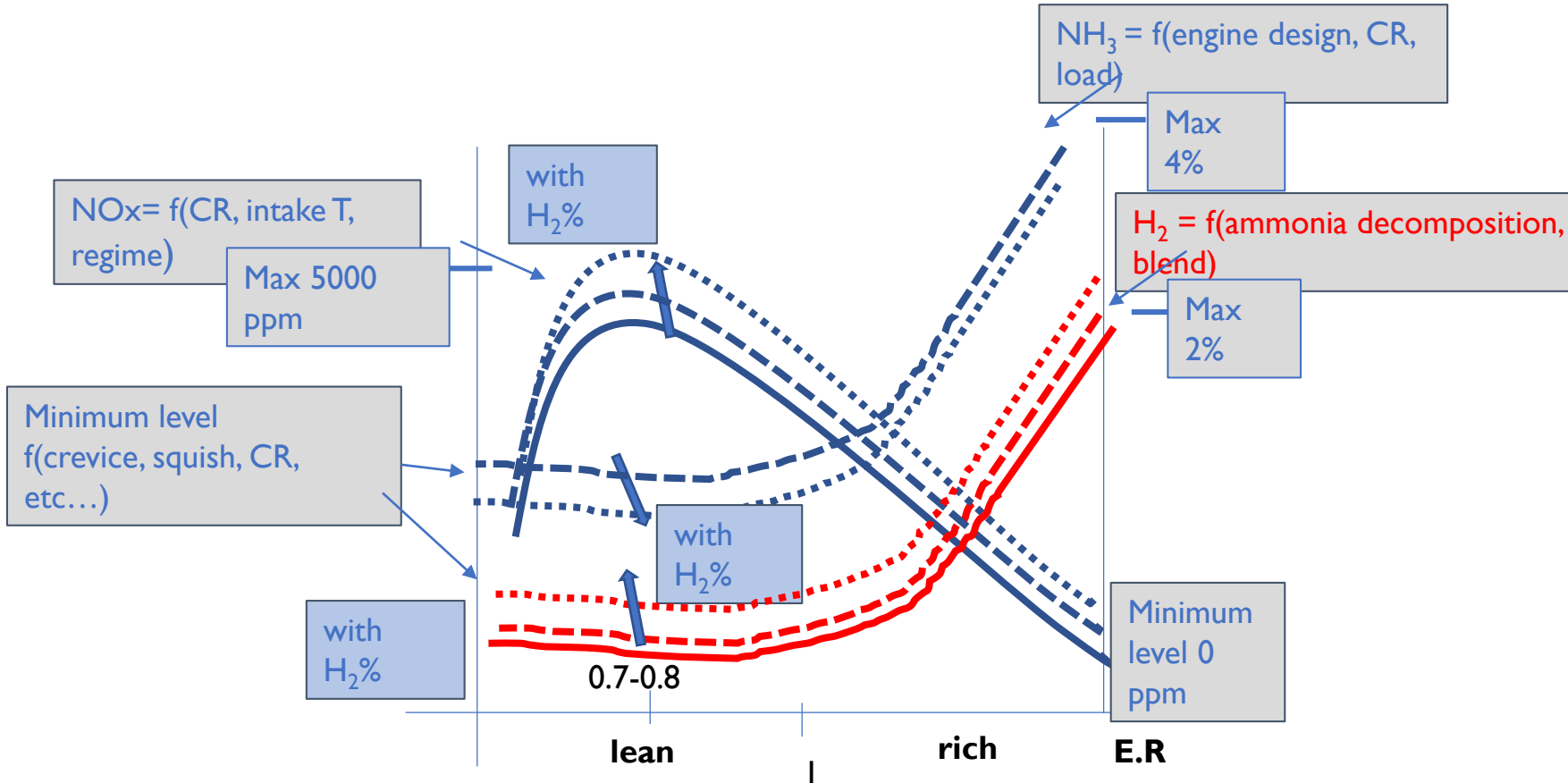


optimised  
window ?

optimised  
window ?



# 3. Consequence on emissions





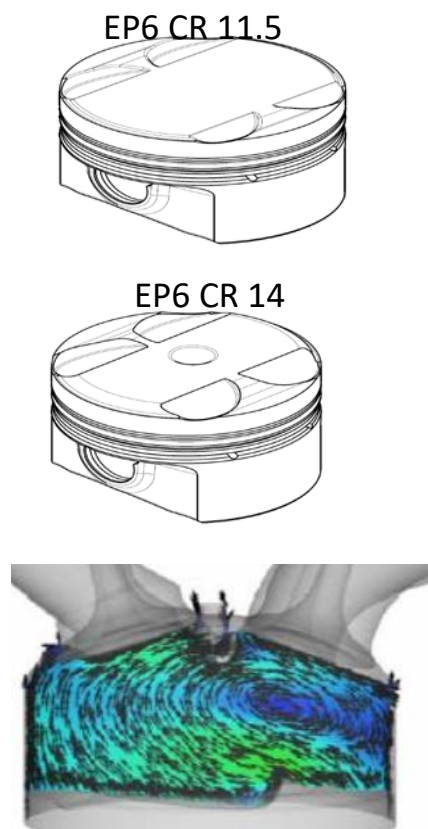
### 3. Consequence on emissions

$$\text{NH}_3 = f(\text{engine design, CR, load})$$

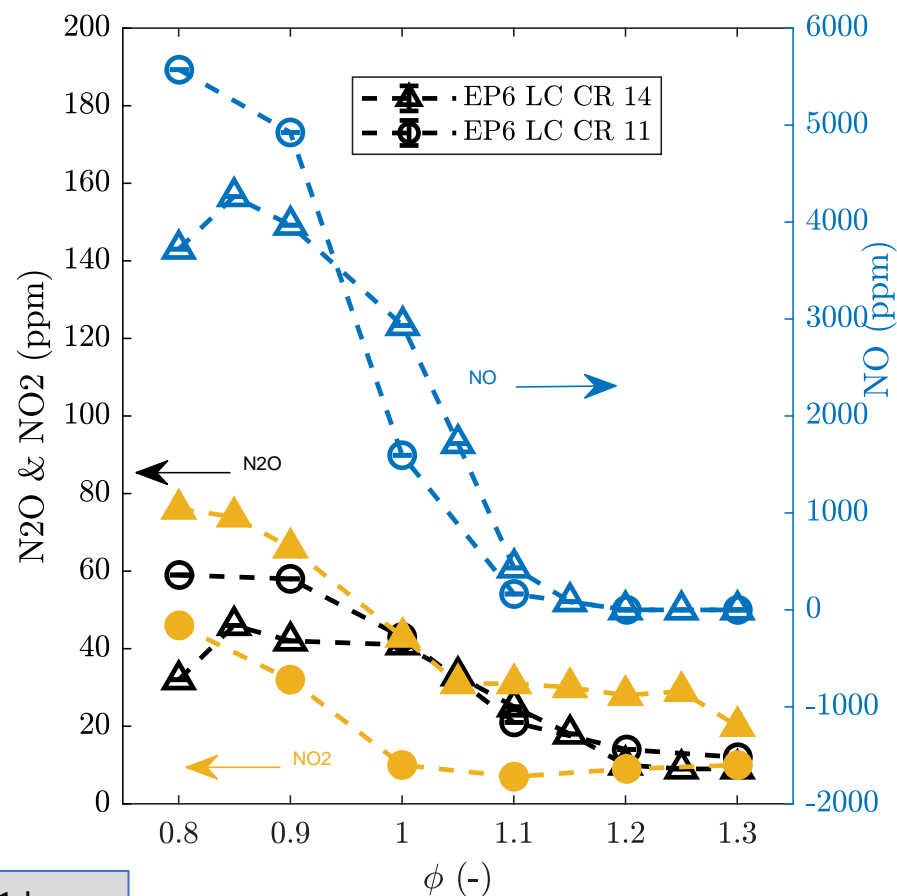
Max

- **NO<sub>x</sub>**
  - Minimum for **rich mixture**, Maximum around 0.7-0.8 until 5000 ppm !
  - Increase with H<sub>2</sub> addition
- **NH<sub>3</sub>**
  - Minimum for **lean mixture/stoichiometry**, max can be 4%
  - **Function of engine design !**
  - **H<sub>2</sub> emissions due to 'in situ' decomposition of NH<sub>3</sub>**

# 3. Consequence on emissions : nitrogen specie



1000 rpm, intake pressure = 1 bar,  
IMEP = 8 b, NH<sub>3</sub> only

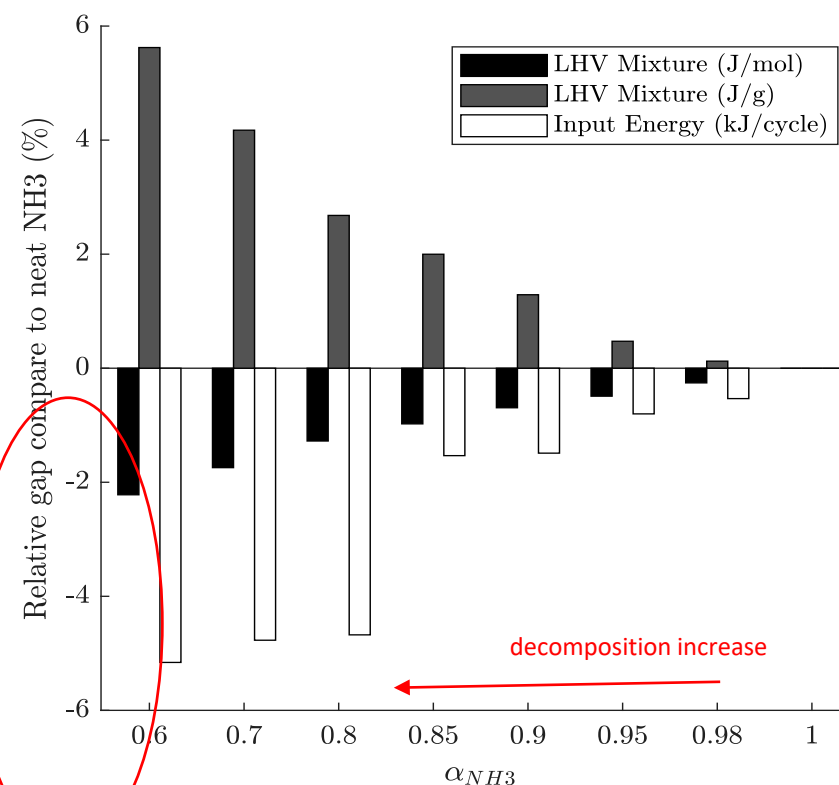
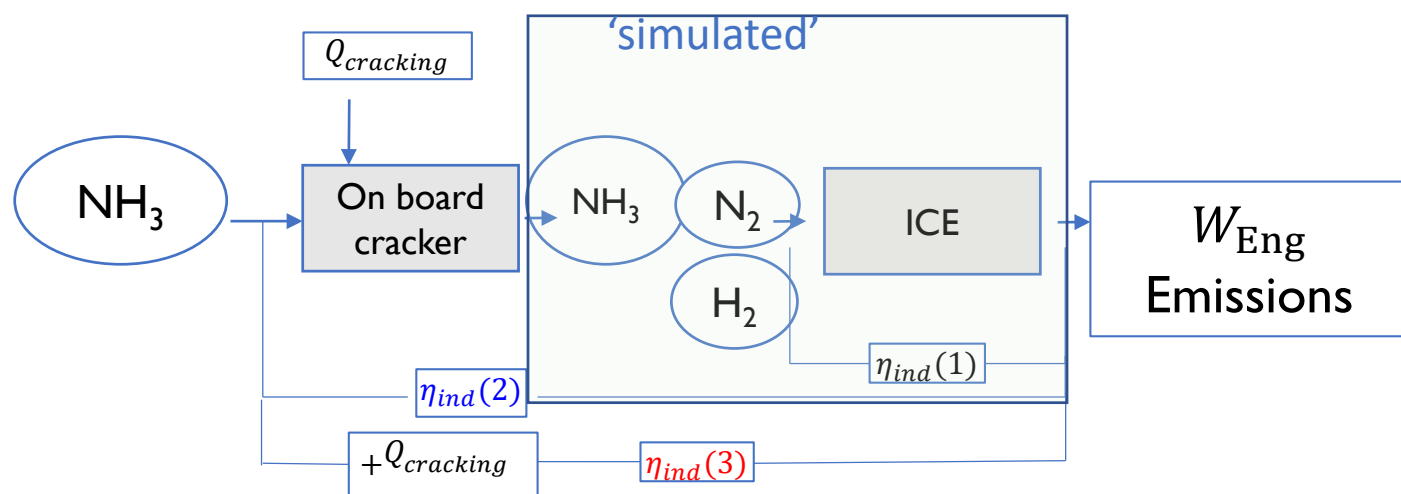
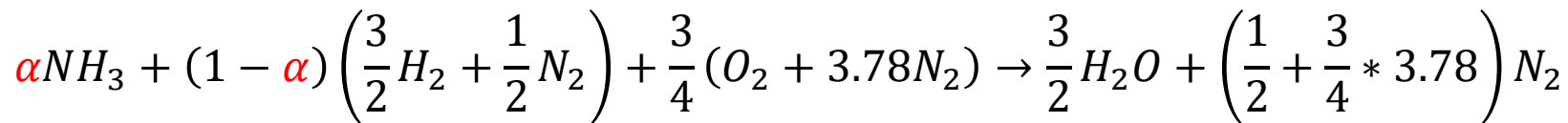


- **NO<sub>2</sub> & N<sub>2</sub>O :**
  - Max for lean, min for rich
- **NO<sub>2</sub>**
  - <<< NO
  - Function of CR ?
- **N<sub>2</sub>O**
  - Max  $\approx$  100 ppm
  - BUT  $\approx$  1.5% CO<sub>2</sub> for GWImpact
  - No real link with CR



areNH3a

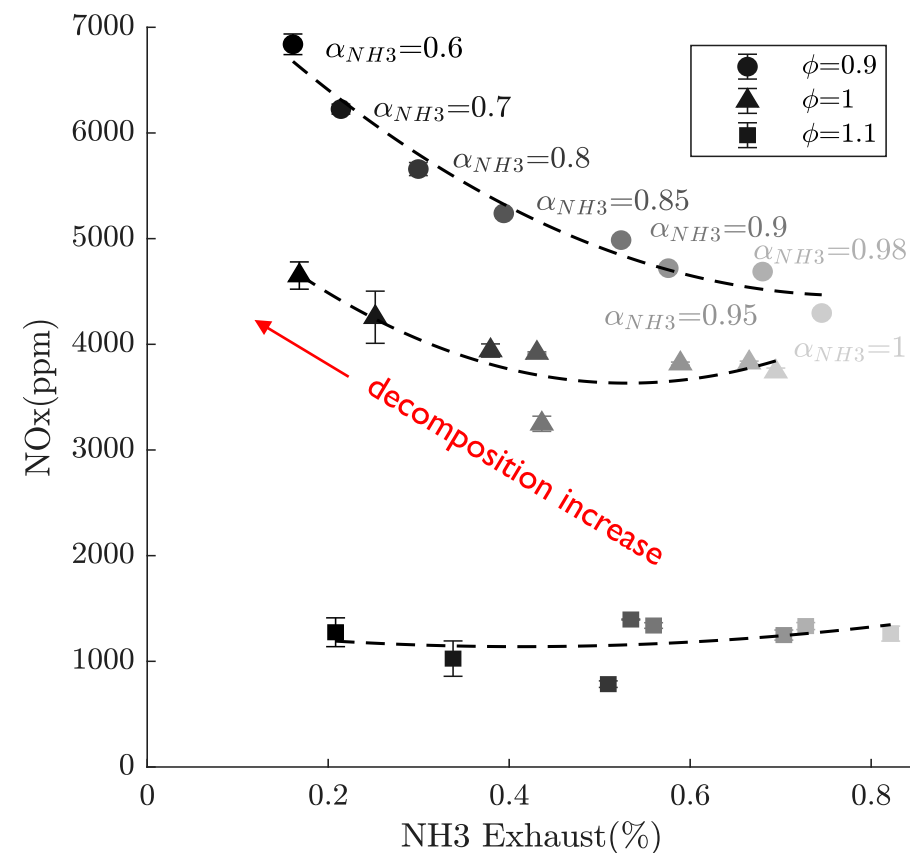
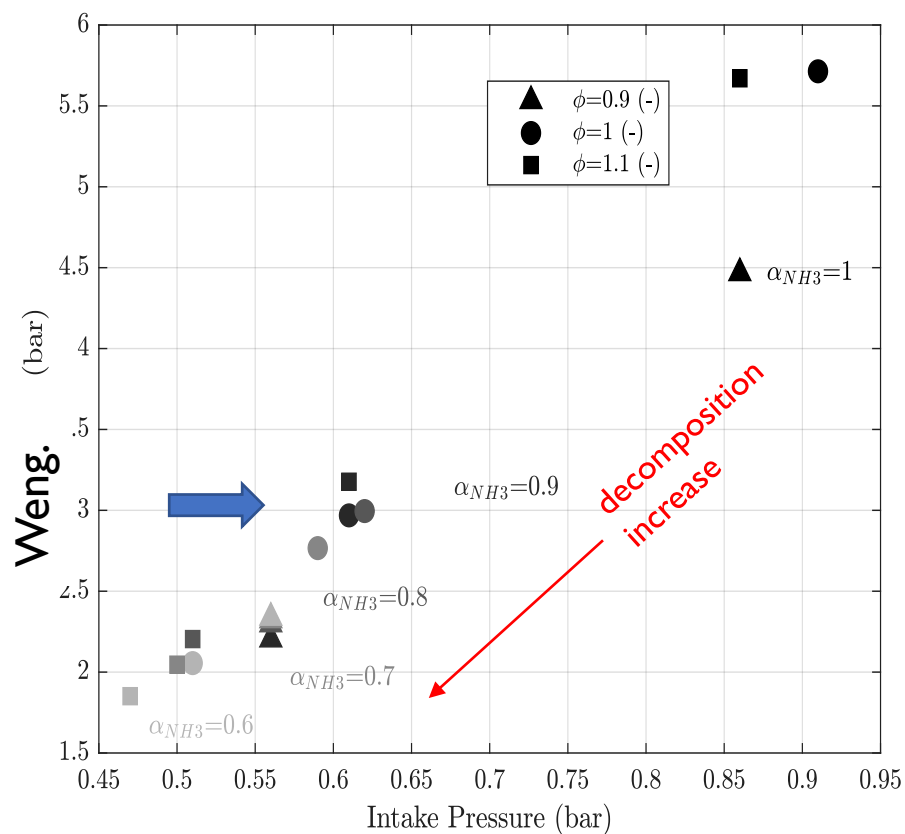
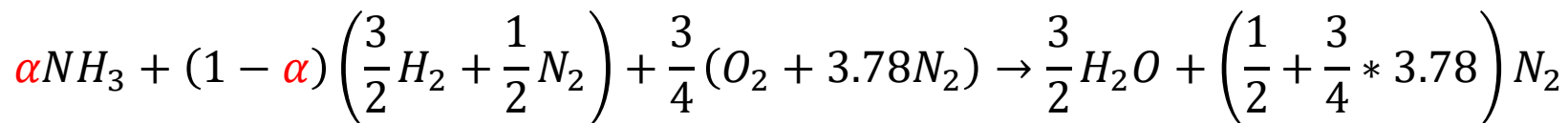
## 4. Ammonia on-board cracking : what benefits ?





areNH3a

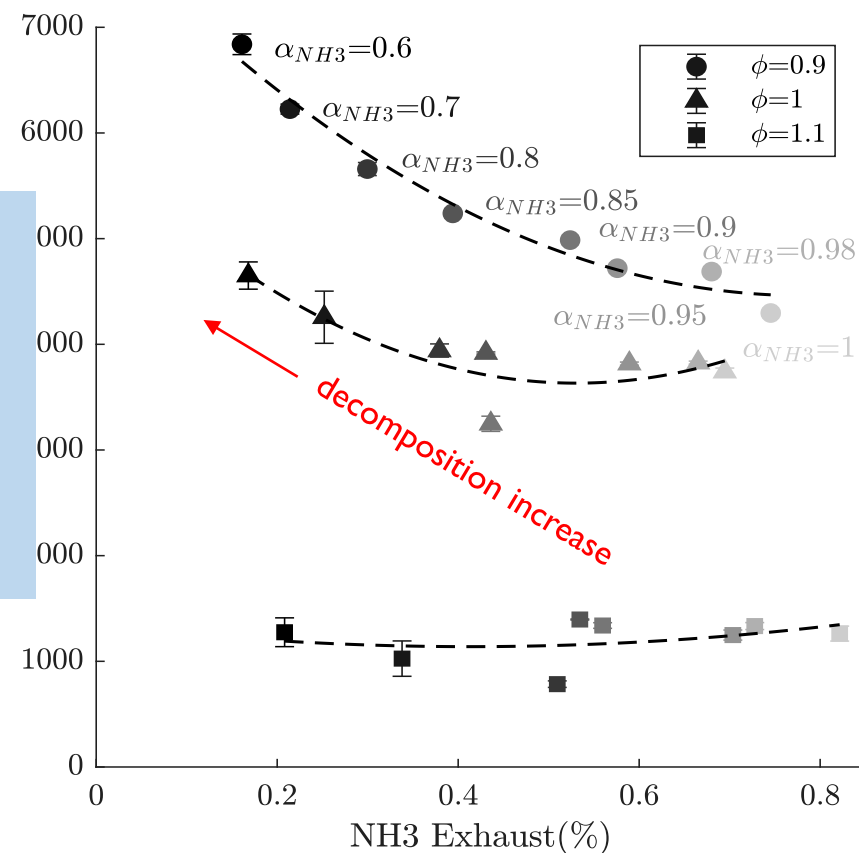
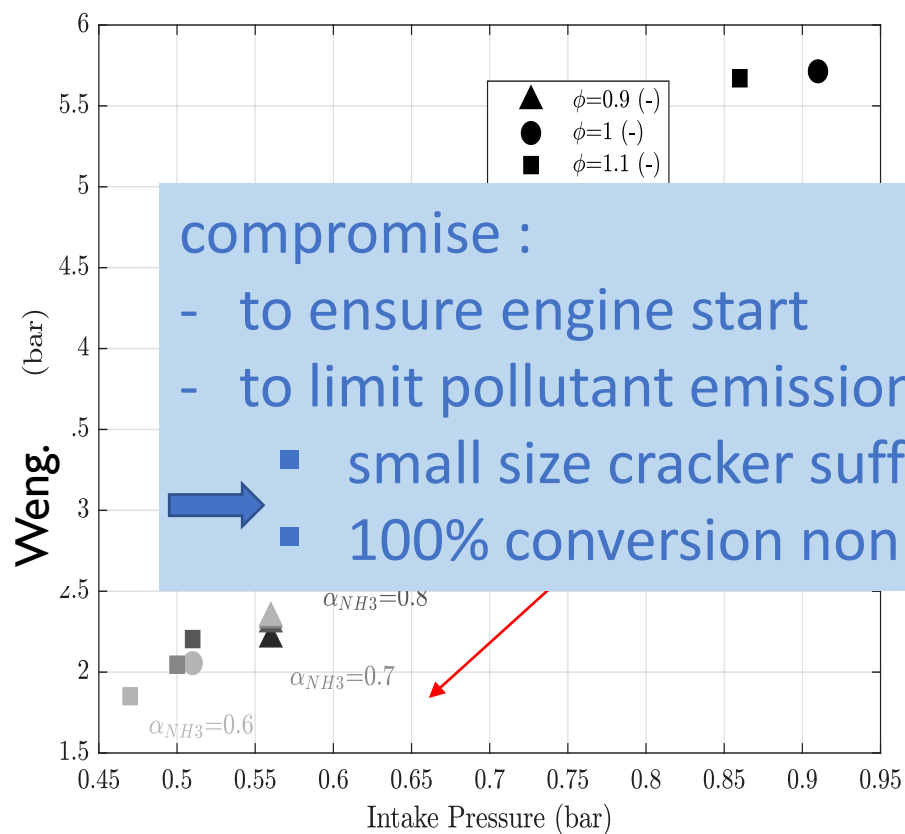
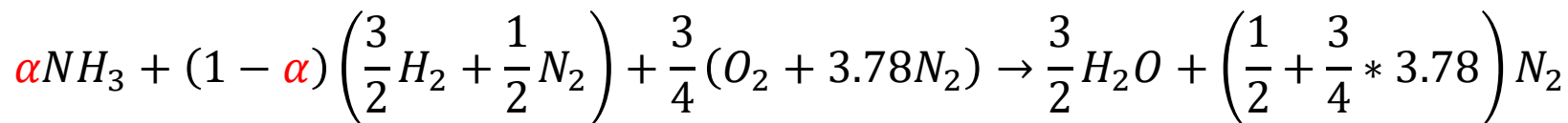
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areNH3a



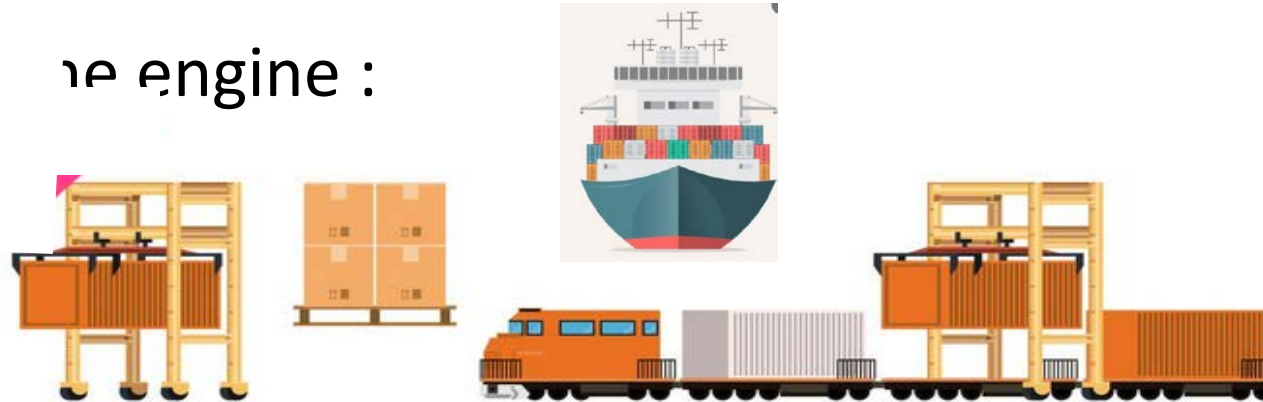
# 5. Conclusions and perspective

## Direct use of ammonia in ICE

### - GOOD NEWS : YES IT IS POSSIBLE !

#### - In standard Gas engine :

- small container
- even with
- means of 'small'



#### - In standard Dies

- addition of s
- more unburn







## Direct use of ammonia for mobility (ICE)

First Workshop ARENHA project, ENGIE Lab CRIGEN, 07-04-2022

*Thank you for your attention*

Website project: <https://arenha.eu/>

Thanks to **STELLANTIS-PRISME** TEAM

J. Bouriot, S. Houillé, C. Dumand, P. Brequigny, B. Raitiere, A. Mercier,



First ARENHA Workshop, ENGIE Lab CRIGEN (April 7<sup>th</sup>, 2022)  
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